# Fluid Dynamics

Conservation Laws I

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## Introduction

In this chapter, we study ideal fluids — fluids that exhibit no viscosity. The discussion applies to both compressible and incompressible flows. Specifically, we examine three fundamental conservation laws:

- Conservation of Mass (Continuity Equation)
- Conservation of Momentum
- Conservation of Energy

## Conservation of Mass

The conservation of mass is expressed by the continuity equation, which in general form reads:

$$\frac{\partial (\textit{density})}{\partial t} + \vec{\nabla} \cdot (\textit{flow}) = \textit{sources} - \textit{sinks}$$

For a fluid with density  $\rho$  and velocity field  $\vec{u}$ , this becomes:

$$\frac{\partial 
ho}{\partial t} + \vec{
abla} \cdot (
ho \vec{u}) = 0$$

**Note:** If mass sources or sinks are present (e.g., chemical reactions or mass injection), the right-hand side is non-zero.

In the case of **incompressible flow** ( $\rho = \text{constant}$ ), the continuity equation simplifies to:

$$\vec{\nabla} \cdot \vec{u} = 0$$

## Conservation of Mass

The divergence  $\vec{\nabla} \cdot \vec{u}$  describes the local rate of volumetric expansion or compression of the fluid:

- $\vec{\nabla} \cdot \vec{u} > 0$ : the flow diverges the fluid expands locally (acts as a source),
- $\vec{\nabla} \cdot \vec{u} < 0$ : the flow converges the fluid compresses locally (acts as a sink).

The conservation of mass can also be written in integral form:

$$-\iiint_{V}rac{\partial
ho}{\partial t}\,d au=\iint_{\partial V}
hoec{u}\cdot dec{a}$$

**Left-hand side:** rate of increase of mass inside the control volume V.

**Right-hand side:** net mass flux *out* of the control volume across the surface  $\partial V$ .

### Conservation of Momentum

The conservation of momentum in a fluid expresses Newton's second law: the rate of change of momentum equals the sum of forces.

For an inviscid fluid, the Euler equation is:

$$\rho\left(\frac{\partial \vec{u}}{\partial t} + (\vec{u} \cdot \vec{\nabla})\vec{u}\right) = -\vec{\nabla}P + \vec{f}_b$$

#### Where:

- Eulerian acceleration total acceleration of a fluid particle,
- pressure gradient force acts from high to low pressure (per unit volume),
- body force density external force per unit volume acting throughout the fluid (e.g., gravity:  $\vec{f_b} = \rho \vec{g}$ ).

## Conservation of Momentum

From the conservation of momentum, we can compute the **total force exerted by pressure** on a control volume:

$$\vec{F}_{\mathsf{pressure}} = - \iint_{\partial V} P \, d\vec{a}$$

This surface integral represents the net pressure force acting on the boundary  $\partial V$ .

Using the divergence theorem:

$$\vec{F}_{\text{pressure}} = -\iiint_V \vec{\nabla} P \, d\tau, \quad \text{since} \quad \iint_{\partial V} P \, d\vec{a} = \iiint_V \vec{\nabla} P \, d\tau$$

**Tip:** For any closed surface,  $\iint_{\partial V} d\vec{a} = 0$ . This allows us to replace a complex surface with a simpler one to ease pressure force calculations, as long as the total surface remains closed.

## Conservation of Energy

The conservation of energy for compressible fluids (e.g., gas) is:

$$\delta Q = dU + P d\tau \quad \Rightarrow \quad \delta q = de + P d\tau$$

while:

$$\delta q = de + P d\tau$$
 with  $e = \frac{1}{\gamma - 1} \frac{P}{\rho}$ 

The conservation of energy leads to the following expressions for specific thermodynamic processes:

- Adiabatic process:  $\delta q = 0 \quad \Rightarrow \quad \frac{P}{\rho^{\gamma}} = \text{const}$  Isothermal process:  $T = \text{const} \quad \Rightarrow \quad \frac{P}{\rho} = \text{const}$
- Polytropic gas:  $\gamma = \frac{C_p}{C_v} \implies P = A\rho^{\gamma_{\text{eff}}}$
- Ideal (isentropic) fluid:  $\Rightarrow \frac{d}{dt} \left( \frac{P}{\rho^{\gamma}} \right) = 0$

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## **Boundary Conditions**

To solve the governing PDEs, we must define the appropriate boundary conditions:

- $\vec{u}_{\perp} = \vec{u}_{\text{surface}}$  (normal velocity must match the boundary velocity),
  - $P = P_{\text{surface}}$  (pressure continuity for ideal fluids).

For static fluids ( $\vec{u} = 0$ ), the boundary conditions reduce to:

- $ec{u}_1 = ec{u}_2$  (velocity continuity across the interface),
- ullet  $P_1=P_2$  (pressure continuity across the interface for ideal fluids).

## Bernoulli's Equation

Bernoulli's equation is derived by requiring the flow to be steady and inviscid. From the momentum equation:

$$\vec{u} \cdot \vec{\nabla} \left( \frac{u^2}{2} + h + \Phi_g \right) = 0 \quad \Rightarrow \quad \frac{u^2}{2} + h + \Phi_g = E$$

where E is constant along each streamline.

Note that Bernoulli's equation holds only if:

$$\vec{u} \cdot \frac{\vec{\nabla}P}{\rho} = \vec{u} \cdot \vec{\nabla}h$$

Special forms:

• Incompressible fluids:  $h = \frac{P}{\rho} \implies \frac{u^2}{2} + \frac{P}{\rho} + \Phi_g = E$ • Ideal gas (isentropic):  $h = \frac{\gamma}{\gamma - 1} \frac{P}{\rho} \implies \frac{u^2}{2} + \frac{\gamma}{\gamma - 1} \frac{P}{\rho} + \Phi_g = E$ 

### Acceleration and Pressure

The dependence of pressure on acceleration is given by the momentum conservation equation:

$$\rho \vec{a} = -\vec{\nabla}P + \vec{f}$$

This simplified form is valid only when external body forces (such as gravity) are negligible.

**Note:** Although  $\vec{\nabla} \times \vec{\nabla} P = 0$  holds identically, the pressure gradient alone cannot generate rotational motion in the fluid. If the flow is rotational, it must be due to other forces (e.g., body forces or viscosity).